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December 19, 2015

Cankaya University
Faculty of Engineering
Mechanical Engineering Department
ME 313 Heat Transfer
Midterm Exam II
Closed Notes Open Book
Fall 2015

- 1) Atmospheric nitrogen at 150°C flows across a long 5-cm diameter circular cylinder at a velocity of 20 m/s. The surface temperature of the cylinder is 500°C . On a per unit length basis, determine the convective heat transfer from the cylinder.



$$T_\infty = 150^\circ\text{C}$$
$$V = 20\text{ m/s}$$
$$T_s = 500^\circ\text{C}$$

$$T_f = \frac{1}{2}(500 + 150) = 325^\circ\text{C} = 598\text{ K} \approx 600\text{ K}$$

$$\rho = 0.5615\text{ kg/m}^3, \quad \mu = 290.5 \times 10^{-7} \frac{\text{N}\cdot\text{s}}{\text{m}^2}$$

$$k = 44.6 \times 10^{-3}\text{ W/m}\cdot\text{K} \quad Pr = 0.701$$

$$Nu_D = C Re_D^n Pr^{1/3}$$

$$Re_D = \frac{\rho V D}{\mu} = 1.94 \times 10^4 \quad \text{so } C = 0.193, n = 0.618$$

$$\bar{h} = \frac{k}{D} Nu_D = 682\text{ W/m}^2\cdot\text{K} \quad Nu_D = 76.6$$

$$\frac{q}{L} = \bar{h} \pi D (T_s - T_\infty) = 3750\text{ W/m}$$

- 2) Water flows at 7.55 kg/s through a 12-cm -diameter ,110-m-long plastic drainage pipe. The water temperature at the pipe inlet is 25 °C, and the ground in which the pipe is buried maintains the temperature of the inside pipe surface at 15 °C.
- find heat transfer coefficient
 - water temperature at the pipe outlet
 - total convective heat transfer

Evaluate properties at $T_{mi} = 25^\circ\text{C}$ We do not know outlet temperature.

$$\rho = 997 \text{ kg/m}^3 \quad c_p = 4183 \text{ J/kg K} \quad \mu = 890.5 \times 10^{-6} \text{ kg/m s}$$

$$V = 89.32 \times 10^{-8} \text{ m}^2/\text{s} \quad k = 0.6071 \text{ W/m K} \quad \underline{\underline{Pr = 6.14}}$$

$$\dot{m} = 7.55 \text{ kg/s} = \rho V A_c \quad A_c = \frac{\pi D^2}{4}$$

$$V = \frac{4\dot{m}}{\rho \pi D^2} = 0.669 \text{ m/s}$$

$$Re_D = \frac{\rho V D}{\mu} = 8.99 \times 10^4 \quad \text{turbulent flow}$$

$$\overline{Nu}_D = \frac{(f/8) [Re_D - 1000] Pr}{1 + 12.7 [\sqrt{f/8}] [Pr^{2/3} - 1]}$$

$$f = \frac{0.25}{\left[\log \left(\frac{5.74}{Re_D^{0.9}} \right) \right]^2} = 0.0183 \quad \leftarrow \text{smooth pipe}$$

$$\overline{Nu}_D = 514$$

$$\overline{h} = \frac{k}{D} \overline{Nu}_D = 2600 \text{ W/m}^2\text{K}$$

$$T_{mo} = T_s - (T_s - T_{mi}) \exp\left[-\frac{PLh}{\dot{m}c_p}\right] = 15.3^\circ\text{C}$$

$$c) \dot{q} = \dot{m}c_p(T_{mo} - T_{mi}) = 3.06 \times 10^5 \text{ W}$$

Since outlet temperature does not differ much from inlet temperature no need to evaluate fluid properties again.

- 3) A flat plate with uniform temperature of 150°C and insulated on one side is suspended vertically in 25°C stagnant atmospheric air. If the plate height and width 40 cm and 1 m, respectively, find the convective heat transfer.



$$T_s = 150^\circ\text{C} \quad T_\infty = 25^\circ\text{C}$$

$$L = 40\text{ cm} \quad W = 1\text{ m}$$

$$T_f = \frac{1}{2}(T_s + T_\infty) = \frac{25 + 150}{2} = 87.5^\circ\text{C} = 361\text{ K}$$

$$\beta = \frac{1}{T_f} = 2.77 \times 10^{-3} \text{ 1/K} \quad \nu = 22.13 \times 10^{-6} \text{ m}^2/\text{s}$$

$$k = 0.0308 \text{ W/m}\cdot\text{K} \quad Pr = 0.696$$

$$Gr_L = \frac{g\beta(T_s - T_\infty)L^3}{\nu^2} = 4.439 \times 10^8$$

$$Ra_L = Gr_L Pr = 3.098 \times 10^8 < 10^9 \text{ flow is laminar}$$

$$\overline{Nu}_L = 0.68 + \frac{0.67 Ra_L^{1/4}}{\left[1 + \left(\frac{0.492}{Pr}\right)^{9/16}\right]^{4/9}} = 68.8$$

$$\overline{h} = \frac{k}{L} \overline{Nu}_L = 5.30 \text{ W/m}^2\cdot\text{K}$$

$$q = FA(T_s - T_\infty) = 265 \text{ W}$$