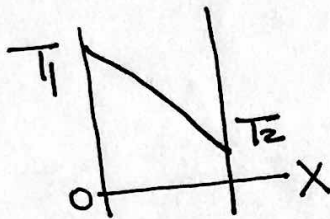


CANKAYA UNIVERSITY  
FACULTY OF ENGINEERING  
MAKINA MUHENDISLIGI BOLUMU  
ME 313 HEAT TRANSFER

Fall 2016

HW 1 SOLUTIONS

- 1) Determine the flux  $q$  and the heat transfer rate across an iron plate with area  $A = 0.5 \text{ m}^2$  and thickness  $L = 0.02 \text{ m}$  [ $k = 70 \text{ W / (m} \cdot \text{°C)}$ ] when one of its surfaces is maintained at  $T_1 = 60 \text{ °C}$  and the other at  $T_2 = 20 \text{ °C}$ .



$$L = 0.02 \text{ m}$$

$$k = 70 \text{ W/m} \cdot \text{°C}$$

$$A = 0.5 \text{ m}^2$$

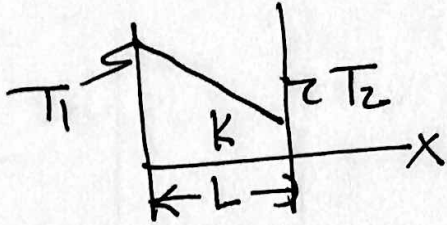
$$T_1 = 60 \text{ °C}$$

$$T_2 = 20 \text{ °C}$$

$$q'' = \frac{k(T_1 - T_2)}{L} = \frac{(70 \text{ W/m} \cdot \text{°C})(60 - 20)}{0.02} = 140 \text{ kW/m}^2$$

$$q = q'' A = (140 \frac{\text{kW}}{\text{m}^2})(0.5 \text{ m}^2) = 70 \text{ kW}$$

- 2) The heat flow rate through a wood board  $L = 2$  cm thick for a temperature difference of  $\Delta T = 25^\circ\text{C}$  between the two surfaces is  $150\text{ W/m}^2$ . Calculate the thermal conductivity of the wood.



$$L = 2\text{ cm}$$

$$\Delta T = T_1 - T_2 = 25^\circ\text{C}$$

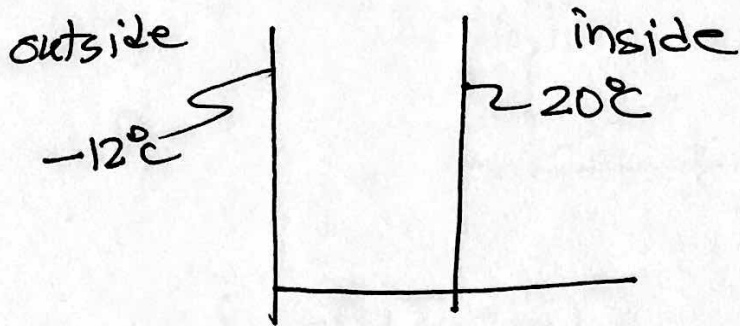
$$q'' = 150\text{ W/m}^2$$

$$q'' = \frac{k \Delta T}{L} = \frac{k (T_1 - T_2)}{L}$$

$$150 \frac{\text{W}}{\text{m}^2} = k \left( \frac{\text{W}}{\text{m}^\circ\text{C}} \right) \left( \frac{25^\circ\text{C}}{0.02\text{m}} \right)$$

$$k = 0.12\text{ W/m}^\circ\text{C}$$

- 3) The inside and outside surface temperatures of a window glass are 20 and  $-12\text{ }^{\circ}\text{C}$ , respectively. If the glass is 80 cm by 40 cm, is 1.6 cm thick and has thermal conductivity  $0.78\text{ W / (m. }^{\circ}\text{C)}$ , determine the heat loss through the glass over 3 h.



$$k = 0.78 \frac{\text{W}}{\text{m}^{\circ}\text{C}}$$

$$L = 1.6\text{ cm} = 0.016\text{ m}$$

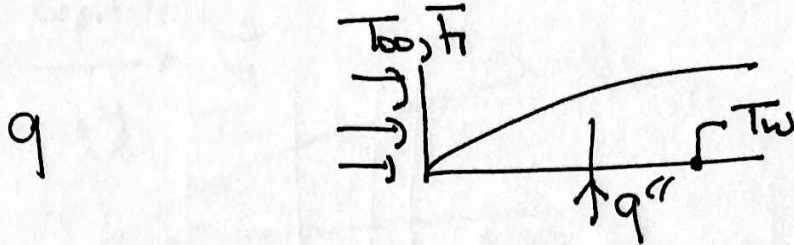
$$A = (0.8)(0.4) \\ = 0.32\text{ m}^2$$

$$q = \frac{kA\Delta T}{L} \\ = \frac{(0.78 \frac{\text{W}}{\text{m}^{\circ}\text{C}})(0.32\text{ m}^2)(20 - (-12))^{\circ}\text{C}}{0.016\text{ m}} = 499.2\text{ W}$$

$$= (499.2 \frac{\text{J}}{\text{s}}) (\frac{3600\text{ s}}{\text{hr}}) (3\text{ hr}) = 5391360\text{ J}$$

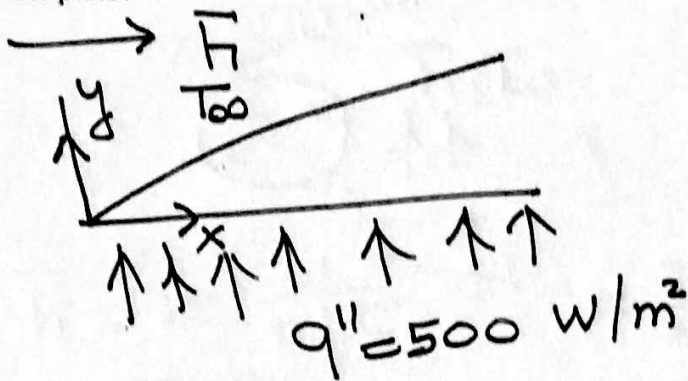
$$= 5391.36\text{ kJ}$$

- 4) An electrically heated plate dissipates heat by convection at a rate of  $q = 8000 \text{ W / m}^2$  into the ambient air  $T_f = 25 \text{ }^\circ\text{C}$ . If the surface of the hot plate is at  $T_w = 125 \text{ }^\circ\text{C}$ , calculate the heat transfer coefficient for convection between the plate and the air.



$$q'' = \bar{h} (T_w - T_\infty)$$
$$8000 \frac{\text{W}}{\text{m}^2} = \bar{h} (125 - 25) \text{ }^\circ\text{C}$$
$$\bar{h} = 80 \text{ W / m}^2\text{ }^\circ\text{C}$$

- 6) Heat is supplied to a plate from its back surface at a rate of  $500 \text{ W / m}^2$  and is removed from its front surface by air flow at  $30^\circ \text{C}$ . If the heat transfer coefficient between the air and the plate surface is  $h = 20 \text{ W / (m}^2 \cdot ^\circ\text{C)}$ , what is the temperature of the front surface of the plate?



$$\bar{h} = 20 \frac{\text{W}}{\text{m}^2 \cdot ^\circ\text{C}}$$

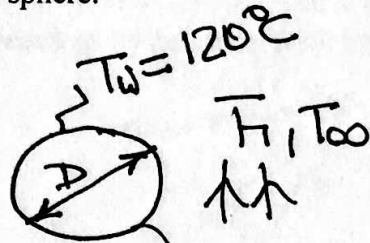
$$T_\infty = 30^\circ\text{C}$$

$$q'' = \bar{h} (T_w - T_\infty)$$

$$500 \frac{\text{W}}{\text{m}^2} = (20 \frac{\text{W}}{\text{m}^2 \cdot ^\circ\text{C}}) (T_w - 30)^\circ\text{C}$$

$$T_w = 55^\circ\text{C}$$

- 5) A 25 cm diameter sphere at 120 °C is suspended in air at 20 °C. If the natural convection heat transfer between the sphere and the air is 15 W / (m<sup>2</sup> · °C), determine the rate of heat loss from the sphere.



$$\bar{h} = 15 \frac{\text{W}}{\text{m}^2 \cdot ^\circ\text{C}} \quad A = \frac{\pi}{4} D^2$$

$$D = 25^\circ\text{C}$$

$$T_\infty = 20^\circ\text{C}$$

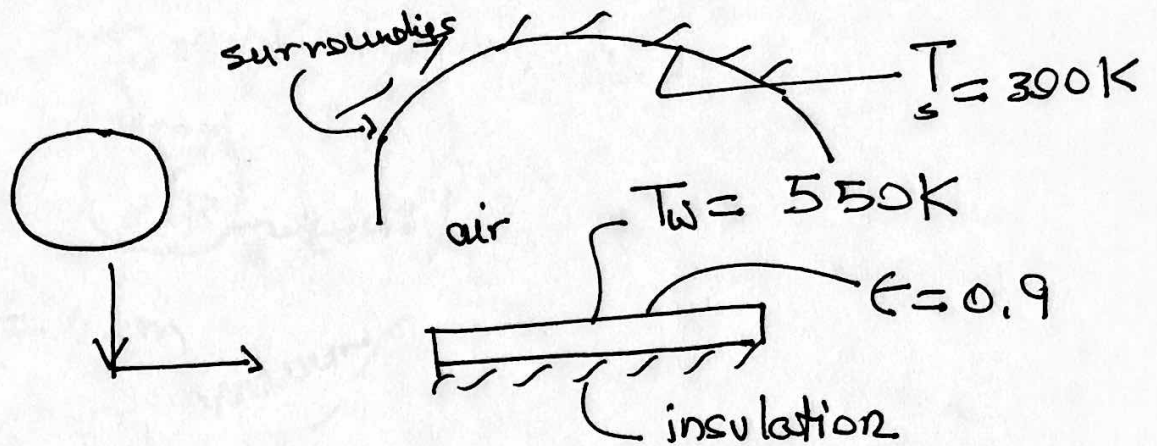
$$q = \bar{h} A (T_w - T_\infty)$$

$$= \left(15 \frac{\text{W}}{\text{m}^2 \cdot ^\circ\text{C}}\right) (\pi) (0.25\text{m})^2 (120 - 20)^\circ\text{C}$$

$$\approx 294 \text{ W}$$

$$= 78 \text{ W}$$

- 7) A heated plate of  $D = 0.2$  m diameter has one of its surfaces insulated, and the other is maintained at  $T_w = 550$  K. If the hot surface has an emissivity  $\epsilon_w = 0.9$  and is exposed to a surrounding area at  $T_s = 300$  K with atmospheric air being the intervening medium, calculate the heat loss by radiation from the hot plate to the surroundings.



1) air is not participating

$$q = A \epsilon \sigma (T_w^4 - T_s^4)$$

$$A = \pi (0.2)^2 = 0.1256 \text{ m}^2$$

$$\sigma = 5.67 \times 10^{-8} \frac{\text{W}}{\text{m}^2 \text{K}^4}$$

$$T_w = 550 \text{ K}$$

$$T_s = 300 \text{ K}$$

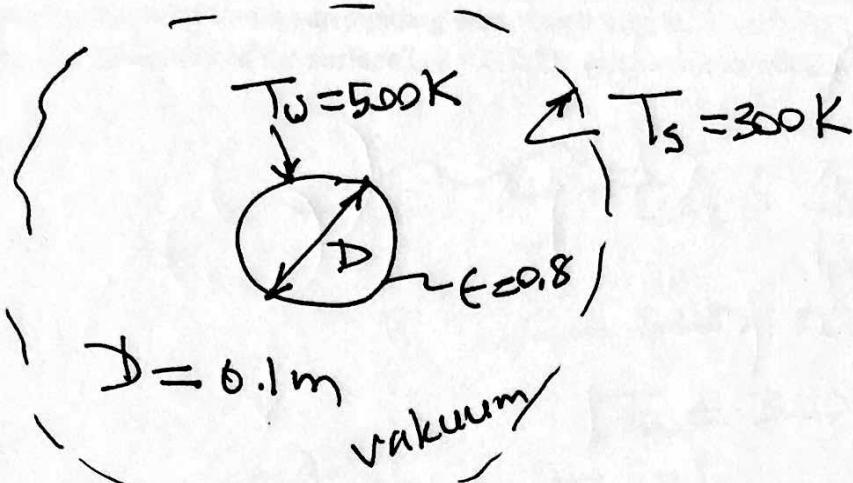
$$q = (0.1256 \text{ m}^2) (0.9) (5.67 \times 10^{-8}) (550^4 - 300^4)$$

$$= 134.5 \text{ W}$$

~~534.58 W~~

(7)

- 8) A sphere 10 cm in diameter is suspended inside a large evacuated chamber whose walls are kept at 300 K. If the surface of the sphere has emissivity  $\epsilon = 0.8$  and is maintained at 500 K, determine the rate of heat loss from the sphere to the walls of the chamber.

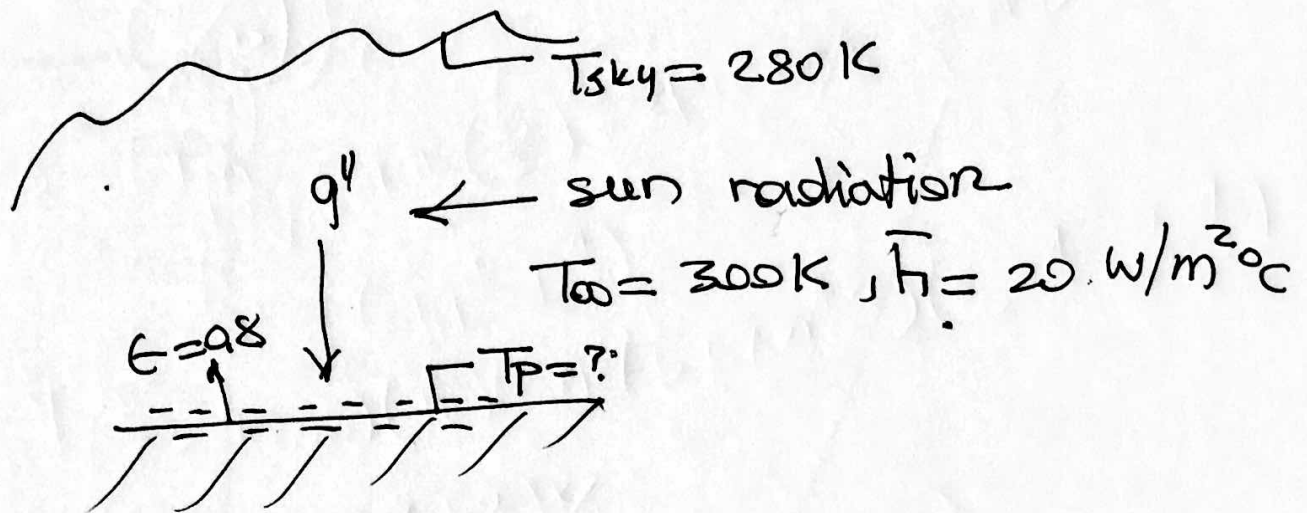


$$A = 4\pi r^2 = 4\pi \left(\frac{D}{2}\right)^2 = 4\pi D^2/4 = \pi D^2$$

$$\begin{aligned} q &= \epsilon A \sigma (T_w^4 - T_s^4) \\ &= (0.8)(\pi)(0.1)^2 (5.67 \times 10^{-8}) (500^4 - 300^4) \\ &= 77.52 \text{ W} \end{aligned}$$



9) A small, thin metal plate of area  $A \text{ m}^2$  is kept insulated on one side and exposed to the sun on the other side. The plate absorbs solar energy at a rate of  $500 \text{ W/m}^2$  and dissipates it by convection into the ambient air at  $T_\infty = 300 \text{ K}$  with a convection heat transfer coefficient  $h_c = 20 \text{ W/(m}^2 \cdot ^\circ\text{C)}$  and by radiation into a surrounding area which may be assumed to be a blackbody at  $T_{\text{sky}} = 280 \text{ K}$ . The emissivity of the surface is  $\epsilon = 0.9$ . Determine the equilibrium temperature of the plate.



$500 \text{ W/m}^2$  absorbed

$$q_c =$$

$$500 = 20(T_p - 300) + (0.9)(567 \times 10^{-8}) \left[ \frac{T_p^4}{5100} - (280)^4 \right]$$

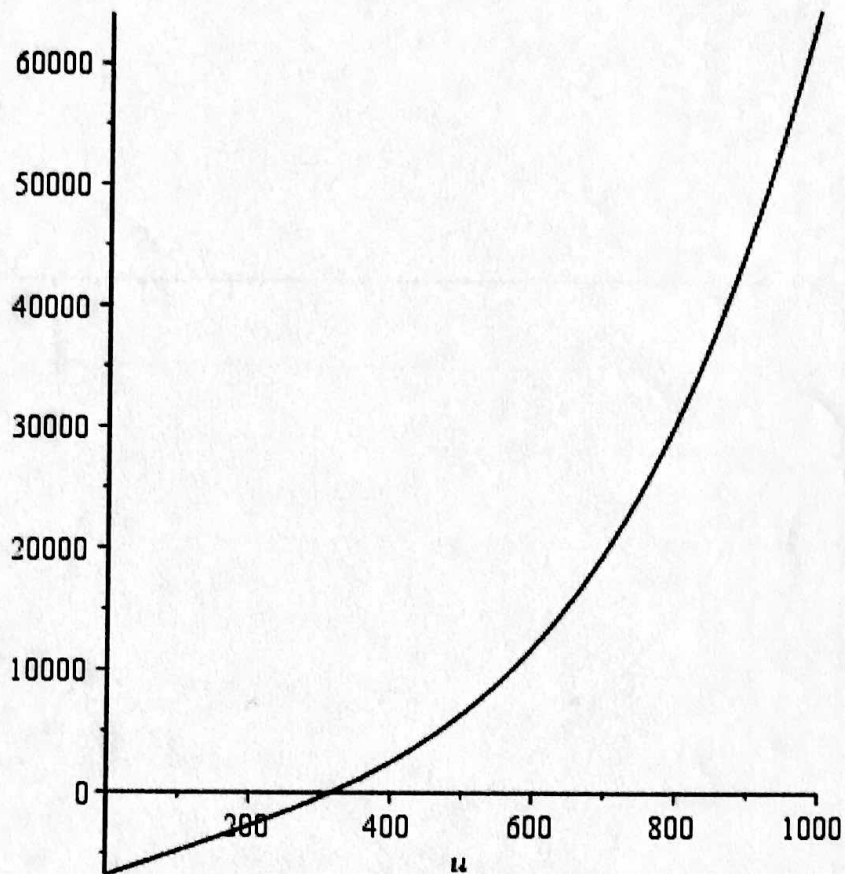
Try Solve using Matlab  
 fzero command  
 or Use Maple 2016  
 see next page

(9)

```

> restart;
>
> # let plate temperature be u
> eq := -500 + 20(u - 300) + (0.9) * (5.67 * 10^-8) * (u^4 - 280^4);
      eq := -6813.658957 + 20 u + 5.103000000 10^-8 u^4
> with(plots) :
>
> plot(eq, u = 0 ..1000);

```



```

> fsolve(eq, u = 200 ..400);
      315.4257948

```

```

>

```

(10)

```
>> f=@(u)-500+20*(u-300)+(0.9)*(5.67e-8)*(u.^4-280^4)
```

```
f =
```

```
@(u)-500+20*(u-300)+(0.9)*(5.67e-8)*(u.^4-280^4)
```

```
>> u=0:10:1000;
```

```
% evaluate function f(u)
```

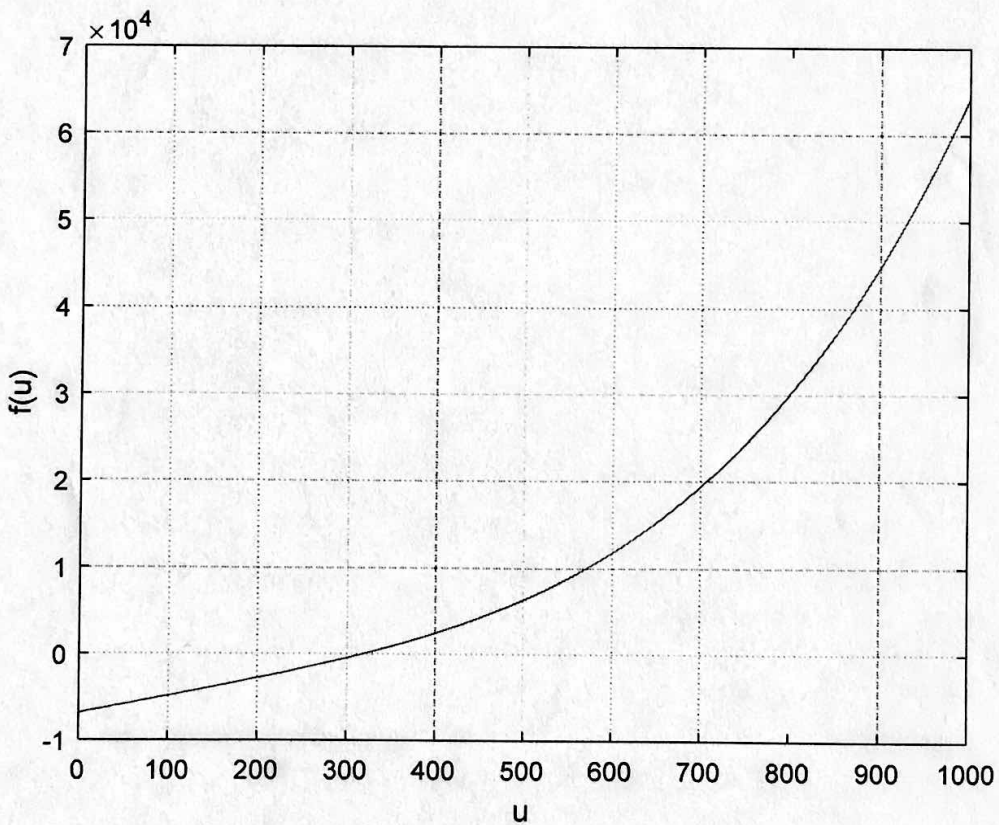
```
>> F=f(u);
```

```
>> plot(u,F)
```

```
>> grid
```

```
>> xlabel('u')
```

```
>> ylabel('f(u)')
```



```
>> fzero(f,300)
```

```
ans =
```

```
315.4258
```

(11)