

Cankaya University
Faculty of Engineering
Mechanical Engineering Department
ME 313 Heat Transfer

Chapter-5 Example Solutions

Fall 2015

- 1) A solid copper sphere of 10-cm diameter [$\rho = 8954 \text{ kg/m}^3$, $c = 383 \text{ J/kg}^\circ\text{C}$, $k = 386 \text{ W/m}^\circ\text{C}$] initially at a uniform temperature $T_i = 250^\circ\text{C}$, is suddenly immersed in a well-stirred fluid which is maintained at a uniform temperature $T_\infty = 50^\circ\text{C}$. The heat transfer coefficient between the sphere and the fluid is $\bar{h} = 200 \text{ W/m}^2^\circ\text{C}$
- (a) Check whether the lumped system analysis is suitable
(b) If it is suitable, determine the temperature of copper sphere at $t=5, 10$ and 20 min after the immersion.

$$\rho = 8954 \text{ kg/m}^3$$

$$c = 383 \text{ J/kg}^\circ\text{C}$$

$$k = 386 \text{ W/m}^\circ\text{C}$$

$$T_i = 250^\circ\text{C} \quad T_\infty = 50^\circ\text{C}$$

$$\bar{h} = 200 \text{ W/m}^2^\circ\text{C}$$

$$L_c = \frac{V}{A_s} = \frac{\frac{4}{3}\pi r_0^3}{4\pi r_0^2} = \frac{r_0}{3} = \frac{D}{6} = \frac{0.1}{6} = 0.01666 \text{ m}$$

$$Bi_c = \frac{\bar{h} L_c}{k} = \frac{(200)(0.01666)}{386} = 8.6 \times 10^{-3}$$

$Bi_c < 0.1$ Lumped model is valid

b)
$$\frac{T(t) - T_\infty}{T_i - T_\infty} = e^{-mt}$$

$$m = (\bar{h} A_s / \rho c V) = 3.5 \times 10^{-3} \text{ s}^{-1}$$

$$T(t) = T_\infty + (T_i - T_\infty) e^{-mt}$$

$$= 50 + 200 e^{-0.0035t}$$

$$t_1 = 300 \text{ s}$$

$$T = 120^\circ\text{C}$$

$$t_2 = 600 \text{ s} \quad T = 74.5^\circ\text{C}$$

$$t_3 = 1200 \text{ s} \quad T = 53^\circ\text{C}$$

2) A solid rod [$\alpha = 2 \times 10^{-5} \text{ m}^2/\text{s}$ and $k = 60 \text{ W}/(\text{m} \cdot ^\circ\text{C})$] of diameter $D = 6 \text{ cm}$, initially at temperature $T_i = 800 \text{ }^\circ\text{C}$, is suddenly dropped into an oil bath at $T_\infty = 50 \text{ }^\circ\text{C}$. The heat transfer coefficient between the fluid and the surface is $h = 400 \text{ W}/(\text{m}^2 \cdot ^\circ\text{C})$.

(a) Using the transient-temperature charts, determine the centerline temperature 10 min after immersion in the fluid.

(b) How long will it take the centerline temperature to reach $100 \text{ }^\circ\text{C}$?

Answer: (a) $54.5 \text{ }^\circ\text{C}$; (b) 5 min 47 s

$$\alpha = 2 \times 10^{-5} \text{ m}^2/\text{s} \quad k = 60 \text{ W}/\text{m}^\circ\text{C} \quad D = 6 \text{ cm} \quad T_i = 800 \text{ }^\circ\text{C}$$

$$T_\infty = 50 \text{ }^\circ\text{C} \quad h = 400 \text{ W}/\text{m}^2 \cdot ^\circ\text{C}$$

a) centerline temperature after 600 s

$$Fo = \frac{\alpha t}{r_0^2} = 13.3$$

$$\frac{1}{Bi} = \frac{k}{hr_0} = \frac{60}{(400)(0.03)} = 5$$

$$\left. \begin{array}{l} Fo = 13.3 \\ \frac{1}{Bi} = 5 \end{array} \right\} \frac{\theta_0}{\theta_i} = \frac{T_0 - T_\infty}{T_i - T_\infty} = 0.006$$

$$T_0 = 50 + 0.006(800 - 50) = 54.5 \text{ }^\circ\text{C}$$

b) How long it will take center temperature to reach $100 \text{ }^\circ\text{C}$?

$$\frac{\theta_0}{\theta_i} = \frac{T_0 - T_\infty}{T_i - T_\infty} = \frac{100 - 50}{800 - 50} = 0.067$$

$$\left. \begin{array}{l} \frac{\theta_0}{\theta_i} = 0.067 \\ \frac{1}{Bi} = 5 \end{array} \right\} Fo = \frac{\alpha t}{r_0^2}$$

$$7.7 = \frac{\alpha t}{r_0^2} = \frac{(2 \times 10^{-5})(t)}{(0.03)^2}$$

$$t = 347 \text{ s}$$

- 3) An orange of diameter 10 cm is initially at a uniform temperature of 30°C . It is placed in a refrigerator in which the air temperature is 2°C . If the transfer coefficient between the air and the surface of the orange is $h=50\text{ W}/(\text{m}^2\cdot^\circ\text{C})$, determine the time required for the center of the orange to reach 10°C . Assume the thermal properties of the orange are the same as those of water at the same temperature. [$\alpha=1.4\times 10^{-7}\text{ m}^2/\text{s}$ and $k=0.59\text{ W}/(\text{m}\cdot^\circ\text{C})$]
 Answer: 1 h 32 min

$$D=10\text{ cm} \quad \alpha=1.4\times 10^{-7}\text{ m}^2/\text{s} \quad k=0.59\text{ W}/\text{m}\cdot^\circ\text{C}$$

$$T_i=30^\circ\text{C} \quad T_\infty=2^\circ\text{C} \quad \bar{h}=50\text{ W}/\text{m}^2\cdot^\circ\text{C}$$

How long it will take for center of orange to reach 10°C ?

$$r_0=0.05\text{ m}$$

$$Bi_c = \frac{\bar{h}L_c}{k} = \frac{\bar{h}r_0}{3k} = \frac{(50)(0.05)}{3(0.59)} = 1.41$$

$Bi > 0.1$ use Heisler charts

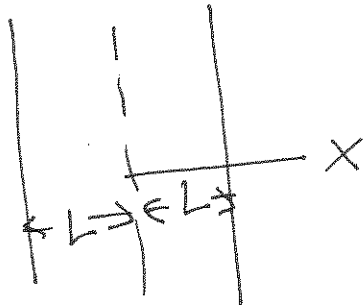
$$\frac{1}{Bi} = \frac{k}{\bar{h}r_0} = \frac{0.59}{(50)(0.05)} = 0.234 \quad \left. \vphantom{\frac{1}{Bi}} \right\} F_0 = 0.31$$

$$\frac{\theta_0}{\theta_i} = \frac{T_0 - T_\infty}{T_i - T_\infty} = \frac{10 - 2}{30 - 2} = 0.286$$

$$F_0 = \frac{\alpha t}{r_0^2}$$

$$0.31 = \frac{(1.4 \times 10^{-7})t}{(0.05)^2} \Rightarrow t = 5535.7\text{ s}$$

4) A steel plate [$\alpha = 1.2 \times 10^{-5} \text{ m}^2/\text{s}$ and $k = 43 \text{ W}/(\text{m} \cdot ^\circ\text{C})$] of thickness $2L = 10 \text{ cm}$, initially at a uniform temperature of 240°C , is suddenly immersed in an oil bath $T_\infty = 40^\circ\text{C}$. The convection heat transfer coefficient between the fluid and the surface is $h = 600 \text{ W}/(\text{m}^2 \cdot ^\circ\text{C})$. How long will it take for the center plane to cool to 100°C ? What fraction of the initial energy is removed during this time?



$$\alpha = 1.2 \times 10^{-5} \text{ m}^2/\text{s} \quad k = 43 \text{ W}/\text{m} \cdot ^\circ\text{C}$$

$$2L = 10 \text{ cm}$$

$$L = 5 \text{ cm} \quad T_i = 240^\circ\text{C}$$

$$h = 600 \frac{\text{W}}{\text{m}^2 \cdot ^\circ\text{C}} \quad T_\infty = 40^\circ\text{C}$$

How long it will take for plate center temperature to cool to 100°C

$$L = \frac{0.1}{2} = 0.05 \text{ m}$$

$$\frac{1}{Bi} = \frac{k}{hL} = \frac{43}{(600)(0.05)} = 1.43$$

$$\frac{\theta_0}{\theta_i} = \frac{T_0 - T_\infty}{T_i - T_\infty} = \frac{100 - 40}{240 - 40} = 0.3$$

$$F_0 = 2.4$$

$$F_0 = \frac{\alpha t}{L^2}$$

$$t = \frac{F_0 L^2}{\alpha}$$

$$t = \frac{(2.4)(0.05)^2}{(1.2 \times 10^{-5})} = 500 \text{ s}$$

Fraction of energy removed during this period

$$Q_0 = \rho c (2L)(A) (T_i - T_\infty) = ?$$

$$= (7833)(457.4)(2 \times 0.05)(240 - 40)$$

$$= 72.847 \text{ MJ}$$

$$\alpha = \frac{k}{\rho c} \Rightarrow c = \frac{k}{\rho \alpha}$$

$$= 457.4 \frac{\text{J}}{\text{kg} \cdot ^\circ\text{C}}$$

$$Bi = \frac{1}{1.43} = 0.7$$

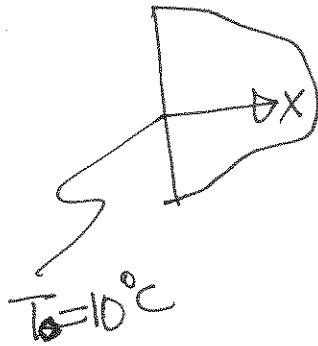
$$Bi^2 F_0 = (0.7)^2 (2.4) = 1.17$$

$$\frac{Q}{Q_0} = 0.77$$

$$Bi = 0.7$$

$$Q = (0.77) Q_0 = 56.1 \text{ MJ}$$

- 5) A thick concrete slab ($\alpha = 7 \times 10^{-7} \text{ m}^2/\text{s}$) is initially at a uniform temperature $T_i = 60^\circ\text{C}$. One of its surfaces is suddenly lowered to 10°C . By treating this as a one-dimensional transient heat conduction problem in a semi-infinite medium, determine the temperatures at depths 5 and 10 cm from the surface 30 min after the surface temperature is lowered.



$$\alpha = 7 \times 10^{-7} \text{ m}^2/\text{s} \quad T_i = 60^\circ\text{C}$$

$$t = 30 \text{ min} = 1800 \text{ s}$$

$$\xi = \frac{x}{2\sqrt{\alpha t}} = \frac{x}{2\sqrt{7 \times 10^{-7} \times 1800}} = 14x$$

$$x = 0.05 \text{ m} \quad \xi = 14 \times 0.05 = 0.7 \quad \text{from figure}$$

$$\frac{T - T_0}{T_i - T_0} = \frac{T - 10}{60 - 10} = 0.67 \rightarrow T = (0.67)(50) + 10 = 43.5^\circ\text{C}$$

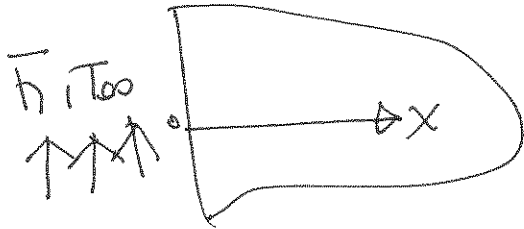
$$x = 0.1 \text{ m} \quad \xi = 14(0.1) = 1.4$$

from figure

$$\frac{T - T_0}{T_i - T_0} = 1.14$$

$$T = 10 + (60 - 10)(1.14) = 57.5^\circ\text{C}$$

- 6) A thick bronze [$\alpha = 0.86 \times 10^{-5} \text{ m}^2/\text{s}$ and $k = 26 \text{ W}/(\text{m} \cdot ^\circ\text{C})$] is initially at a uniform temperature 250°C . Suddenly the surface is exposed to a coolant 25°C . Assuming that the heat transfer coefficient for convection between the fluid and the surface is $150 \text{ W}/(\text{m}^2 \cdot ^\circ\text{C})$, determine the temperature 5 cm from the surface 10 min after the exposure.
 Answer: 205°C



$$\alpha = 0.86 \times 10^{-5} \text{ m}^2/\text{s}$$

$$k = 26 \frac{\text{W}}{\text{m} \cdot ^\circ\text{C}}$$

$$T_i = 250^\circ\text{C}$$

$$T_\infty = 25^\circ\text{C}$$

$$h = 150 \frac{\text{W}}{\text{m}^2 \cdot ^\circ\text{C}}$$

$$t = 10 \text{ min} = 600 \text{ s}$$

$$\xi = \frac{x}{2\sqrt{\alpha t}} = \frac{0.05}{2\sqrt{(0.86 \times 10^{-5}) \times (600)}} = 0.35$$

$$\frac{h\sqrt{\alpha t}}{k} = \frac{150\sqrt{(0.86 \times 10^{-5}) \times (600)}}{26} = 0.41$$

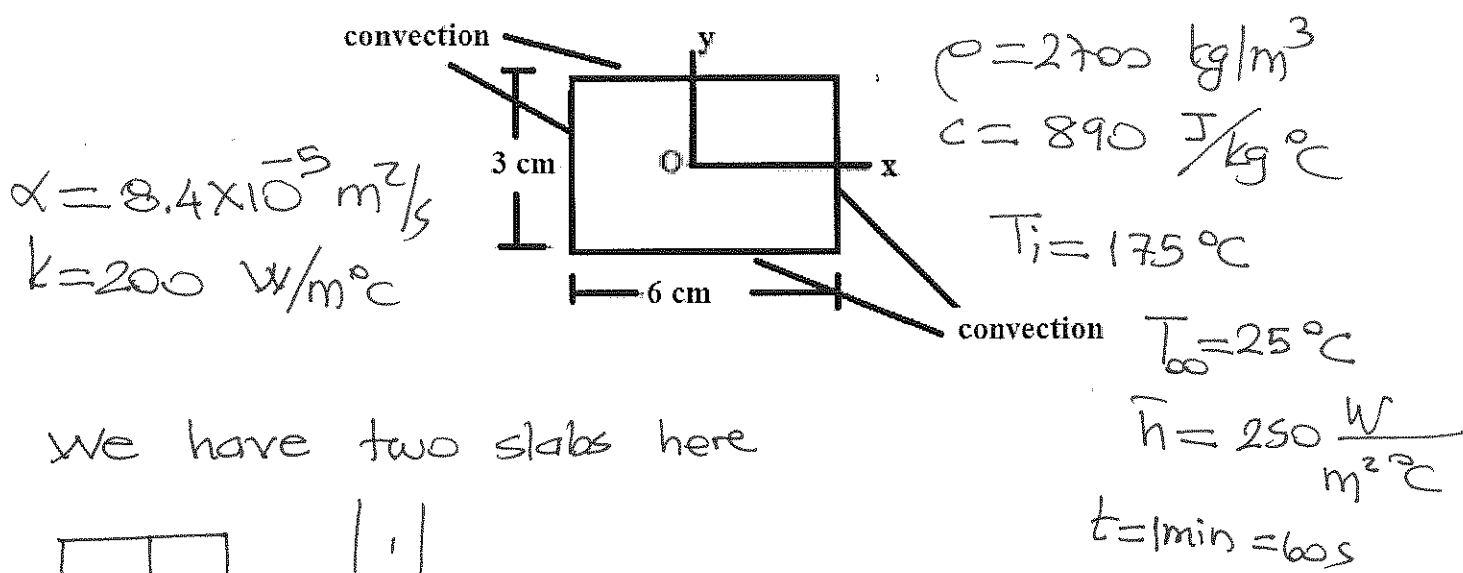
$$1 - \frac{T - T_\infty}{T_i - T_\infty} = 0.2$$

$$1 - \frac{T - T_\infty}{T_i - T_\infty} = 0.2$$

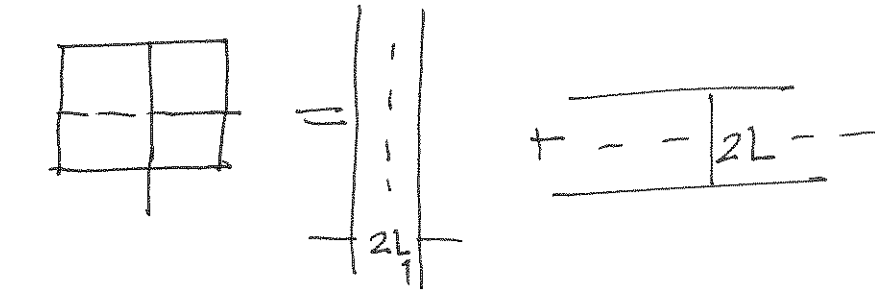
$$\frac{T - T_\infty}{T_i - T_\infty} = 1 - 0.2 = 0.8$$

$$T = 25 + 0.8(250 - 25) = 205^\circ\text{C}$$

7) A rectangular aluminum bar 6 cm by 3 cm [$k=200 \text{ W/(m}\cdot\text{°C)}$, $c_p=890 \text{ J/(kg}\cdot\text{°C)}$, $\rho=2700 \text{ kg/m}^3$, and $\alpha=8.4 \times 10^{-5} \text{ m}^2/\text{s}$] is initially at a uniform temperature $T_i=175 \text{ °C}$. Suddenly the surfaces are subjected to convective cooling with a heat transfer coefficient $h=250 \text{ W/(m}^2\cdot\text{°C)}$ into an ambient at $T_\infty=25 \text{ °C}$ as shown in figure. Determine the center temperature T_0 of the bar $t=1 \text{ min}$ after the start of the cooling.
 Answer: 107.5 °C



We have two slabs here



First slab

$$L_1 = \frac{6}{2} = 3 \text{ cm} = 0.03 \text{ m}$$

$$(Fo)_1 = \frac{\alpha t}{L_1^2} = \frac{(8.4 \times 10^{-5})(60)}{(0.03)^2} = 5.6 \quad \left. \begin{array}{l} \\ \end{array} \right) \frac{\theta_0}{\theta_i} = 0.85$$

$$\frac{1}{Bi} = \frac{k}{hL_1} = \frac{200}{(250)(0.03)} = 26.7$$

$\therefore \left(\frac{\theta_0}{\theta_i} \right)_1 = 0.85$ for first slab

Second slab:

$$L_2 = \frac{3}{2} = 1.5 \text{ cm} = 0.015 \text{ m}$$

$$(Fo)_2 = \frac{\alpha t}{L_2^2} = \frac{(8.4 \times 10^{-5})(60)}{(0.015)^2} = 22.4 \quad \left. \begin{array}{l} \\ \end{array} \right) \left(\frac{\theta_0}{\theta_i} \right)_2 = 0.65$$

$$\frac{1}{Bi} = \frac{k}{hL_2} = \frac{200}{(250)(0.015)} = 53.3$$

$$\left(\frac{\theta_0}{\theta_i}\right)_2 = 0.65 \quad \text{for the second slab}$$

Hence for the product

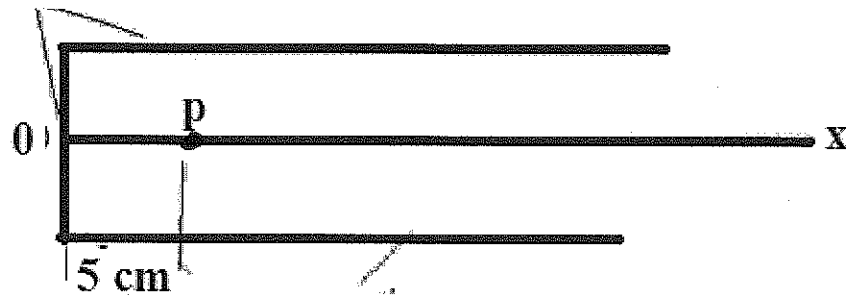
$$\begin{aligned}\left(\frac{\theta_0}{\theta_i}\right) &= \left(\frac{\theta_0}{\theta_i}\right)_1 \left(\frac{\theta_0}{\theta_i}\right)_2 \\ &= (0.89)(0.65) = 0.55\end{aligned}$$

$$\begin{aligned}0.55 &= \frac{T_0 - T_\infty}{T_i - T_\infty} \Rightarrow T_0 = T_\infty + 0.55(T_i - T_\infty) \\ &= 25 + 0.55(175 - 25) \\ &= 107.5^\circ\text{C}\end{aligned}$$

8)

A semi-infinite strip, $0 < x < \infty$, $0 < y < 10$ cm, of fireclay brick [$k=1$ W/(m·°C), and $\alpha = 5.4 \times 10^{-7}$ m²/s] is initially at a uniform temperature $T_i = 340$ °C. Suddenly all surfaces are subjected to convection, with a heat transfer coefficient $h=100$ W/(m²·°C) into an ambient at $T_\infty = 40$ °C. Calculate the temperature T_0 of a point P located along the midplane at a distance $L=5$ cm from the surface as shown in figure, $t=2$ h after the start of the cooling.

convection



convection

$$\alpha = 5.4 \times 10^{-7} \text{ m}^2/\text{s}$$

$$k = 1 \text{ W/m}^\circ\text{C}$$

Answer: 51.4 °C

$$T_i = 340^\circ\text{C}$$

$$T_\infty = 40^\circ\text{C}$$

$$\bar{h} = 100 \text{ W/m}^2^\circ\text{C}$$

$$t = 2 \text{ hrs}$$

Point P:

$$x = 5 \text{ cm}$$

$$y = 5 \text{ cm}$$

1) We have a slab of thickness $2L = 10$ cm subjected to convection

$$L = 10/2 = 0.05 \text{ m}$$

$$Fo = \frac{\alpha t}{L^2} = \frac{5.4 \times 10^{-7} \times 7200}{(0.05)^2} = 1.56 \quad \left. \vphantom{Fo} \right\} \left(\frac{\theta}{\theta_0} \right) = 0.095$$

$$\frac{1}{Bi} = \frac{k}{hL} = \frac{1}{(100)(0.05)} = 0.2$$

2) We have a semi-infinite body subjected to convection

$$\xi = \frac{x}{2\sqrt{\alpha t}} = \frac{0.05}{2\sqrt{5.4 \times 10^{-7} (7200)}} = 0.4$$

$$\frac{\bar{h} \sqrt{\pi t}}{k} = \frac{100 \sqrt{5.4 \times 10^{-7}} (7200)}{k} = 6.2$$

$$\frac{T - T_{\infty}}{T_i - T_{\infty}} = 0.6 \Rightarrow \left(\frac{T - T_{\infty}}{T_i - T_{\infty}} \right)_2 = 0.4$$

or $\left(\frac{\theta}{\theta_i} \right)_2 = 0.4$

$$\left(\frac{\theta}{\theta_i} \right)_2 = 0.4$$

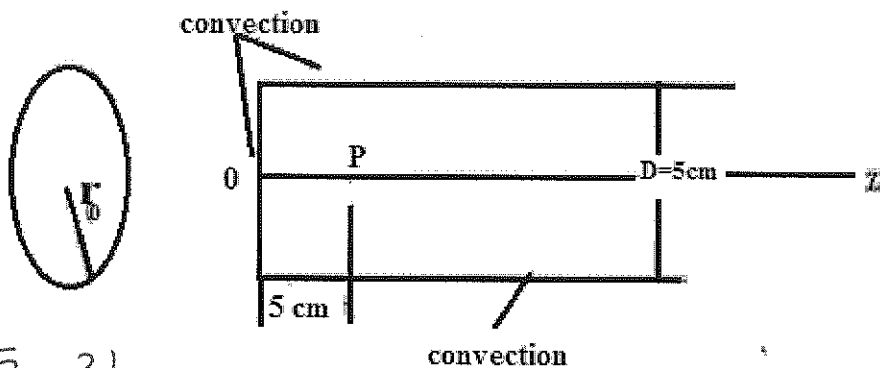
Product solution

$$\left(\frac{\theta}{\theta_i} \right) = \left(\frac{\theta}{\theta_i} \right)_2 \left(\frac{\theta}{\theta_i} \right)_1 = (0.4)(0.095) = 0.038$$

$$T = T_{\infty} + 0.038(T_i - T_{\infty})$$

$$= 40 + 0.038(340 - 40) = 51.4^{\circ}\text{C}$$

- 9) A semi-infinite, cylindrical iron bar [$\alpha = 1.6 \times 10^{-5} \text{ m}^2/\text{s}$ and $k = 60 \text{ W}/(\text{m} \cdot ^\circ\text{C})$] of diameter $D = 5 \text{ cm}$, confined to the region $0 \leq x < \infty$, is initially at a uniform temperature $T_i = 330^\circ\text{C}$. Suddenly the surfaces are subjected to convection with a heat transfer coefficient $h = 200 \text{ W}/(\text{m}^2 \cdot ^\circ\text{C})$ into an ambient at $T_\infty = 30^\circ\text{C}$. Determine the temperature T_0 of a point P located along the axis $L = 3 \text{ cm}$ from the flat surface $t = 2 \text{ min}$ after the start of the cooling. See figure given below.



$$\alpha = 1.6 \times 10^{-5} \text{ m}^2/\text{s}$$

$$k = 60 \text{ W}/\text{m}^\circ\text{C}$$

$$D = 5 \text{ cm}$$

$$T_i = 330^\circ\text{C} \quad h = 200 \text{ W}/\text{m}^2\text{C}$$

$$t = 2 \text{ min} = 60 \text{ s} \times 2 = 120 \text{ s}$$

1) infinite cylinder

$$r_0 = 2.5 \text{ cm} = 0.025 \text{ m}$$

$$Fo = \frac{\alpha t}{r_0^2} = \frac{(1.6 \times 10^{-5})(120)}{(0.025)^2} = 3.1$$

$$\frac{1}{Bi} = \frac{k}{hr_0} = \frac{60}{(200)(0.025)} = 12$$

$$\left(\frac{\theta}{\theta_0}\right)_1 = 0.62$$

2) semi-infinite body

$$\xi = \frac{x}{2\sqrt{\alpha t}} = \frac{0.03}{2\sqrt{(1.6 \times 10^{-5})(120)}} = 0.34$$

$$1 - \frac{T - T_\infty}{T_i - T_\infty} = 0.08$$

$$\frac{h\sqrt{\alpha t}}{k} = \frac{200\sqrt{(1.6 \times 10^{-5})(120)}}{60} = 0.15$$

$$\left(\frac{\theta}{\theta_i}\right)_2 = 1 - 0.08 = 0.92$$

$$\left(\frac{\theta}{\theta_i}\right) = \left(\frac{\theta}{\theta_i}\right)_1 \left(\frac{\theta}{\theta_i}\right)_2 = 0.62(0.92) = 0.57$$

$$(T - T_\infty)/(T_i - T_\infty) = 0.57 \rightarrow T = 201^\circ\text{C}$$