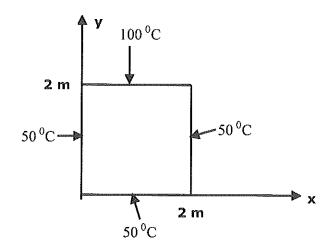
# CANKAYA UNIVERSITY FACULTY OF ENGINEERING MECHANICAL ENGINEERING DEPARTMENT ME 313 HEAT TRANSFER

### **CHAPTER-4**

# **EXAMPLE SOLUTIONS**

# 1. Consider two dimensional problem show below,



Consider rectangular plate given in the figure. Calculate the temperature at the midpoint of the plate for the case in which x=1 m, y=1 m.

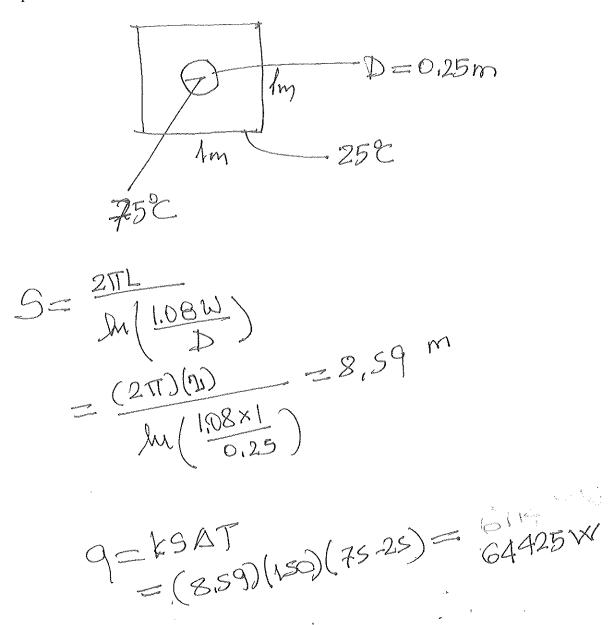
$$\frac{T-T_{1}}{T_{2}-T_{1}} = \left(\frac{2}{T_{1}}\right)\left(\frac{2}{T_{1}}\right)\left(\frac{Sirth(\frac{T}{2})}{Sirth(\frac{T}{2})}\right)Sin(\frac{T}{2})$$

$$+ \frac{2}{T_{1}}\left(\frac{2}{3}\right)\left(\frac{Sirth(\frac{T}{2})}{Sinh(\frac{3T}{2})}\right)Sin(\frac{3T}{2}) + \dots$$

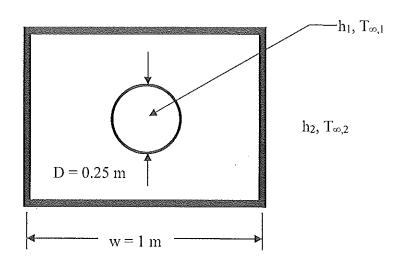
$$= 0.2499$$

$$T = 50 + 0.25(50) = 62.5 \text{ °C}.$$

2. A hole of diameter D = 0.25 m is drilled through the center of a solid block of square cross section with w = 1 m on a side. The hole is drilled along length l = 2 m of block, which has thermal conductivity k = 150 W/m.K. A warm fluid passing through the hole maintains an inner surface temperature of  $T_1 = 75^{\circ}$ C, while the outer surface of the block is kept at  $T_2 = 25^{\circ}$ C. What is the rate of heat transfer through the block?



3. A hole of diameter D = 0.25 m is drilled through the center of a solid block of square cross section with w = 1 m on a side. The hole is drilled along the length, l = 2 m, of the block, which has a thermal conductivity of k = 150 W/m.K. The outer surfaces are exposed to ambient air, with  $T_{\infty,2} = 25$  °C and  $h_2 = 4$  W/m<sup>2</sup>.K, while hot oil flowing through the hole is characterized by  $T_{\infty,1} = 300$  °C and  $h_1 = 50$  W/m<sup>2</sup>.K. Determine the corresponding heat rate and surface temperatures.

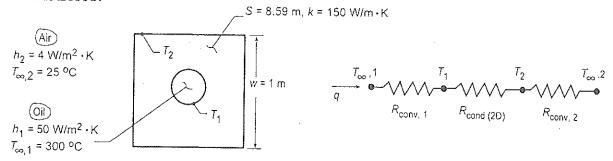


see next page for solution 3

KNOWN: Dimensions, shape factor, and thermal conductivity of square rod with drilled interior hole. Interior and exterior convection conditions.

FIND: Heat rate and surface temperatures.

## SCHEMATIC:



ASSUMPTIONS: (1) Steady-state, two-dimensional conduction, (2) Constant properties, (3) Uniform convection coefficients at inner and outer surfaces.

ANALYSIS: The heat loss can be expressed as

$$q = \frac{T_{\infty,1} - T_{\infty,2}}{R_{conv,1} + R_{cond(2D)} + R_{conv,2}}$$

where

$$R_{conv,1} = (h_1 \pi D_1 L)^{-1} = (50 \text{ W/m}^2 \cdot \text{K} \times \pi \times 0.25 \text{ m} \times 2 \text{ m})^{-1} = 0.01273 \text{ K/W}$$

$$R_{cond(2D)} = (Sk)^{-1} = (8.59 \text{ m} \times 150 \text{ W/m} \cdot \text{K})^{-1} = 0.00078 \text{ K/W}$$

$$R_{conv,2} = (h_2 \times 4wL)^{-1} = (4W/m^2 \cdot K \times 4m \times 1m)^{-1} = 0.0625K/W$$
.

Hence,

$$q = \frac{(300 - 25)^{\circ} C}{0.076 K/W} = 3.62 kW$$

$$T_1 = T_{\infty,1} - qR_{conv,1} = 300^{\circ}C - 46^{\circ}C = 254^{\circ}C$$

$$T_2 = T_{\infty,2} + qR_{conv,2} = 25^{\circ}C + 226^{\circ}C = 251^{\circ}C$$

COMMENTS: The largest resistance is associated with convection at the outer surface, and the conduction resistance is much smaller than both convection resistances. Hence,  $(T_2 - T_{\infty,2}) > (T_{\infty,1} - T_1) > (T_1 - T_2)$ .